

CITY OF WINNIPEG

Pan Am Pool - 25 Poseidon Bay Solar Energy Opportunities Reassessment

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EXECUTIVE SUMMARY

KGS Group was retained by the City of Winnipeg to update the previous 2018 study of solar energy opportunities for the Pan Am Pool. The intent of implementing solar energy opportunities is to reduce the operating costs of the facility and reduce Greenhouse Gas (GHG) emissions, while taking advantage of current Provincial and Federal Government funding opportunities. This report builds on the previous report, revisiting the proposed solar ventilation air heating systems and providing recommendations and opinions on probable costs for replacing five of the building's original air handling units (F-1 to F-5).

The Pan Am Pool does not currently utilize any solar, geothermal, or any other non-traditional source of energy. All energy is currently sourced from natural gas and electricity. The building provides a good opportunity for solar energy capture because of its large South facing wall.

Two options for solar ventilation air heating are considered. A solar wall covering the majority of the main pool area's south facing elevation offers a considerable saving in Greenhouse Gas (GHG) emissions of 137.0 tC02 per year, at an estimate capital cost of \$630,000 resulting in a payback period of 22 years. Solar ducts installed on the lap pool addition's roof offer a 47.5 tCO2 reduction in annual GHG emissions at an estimated capital cost of \$198,000 resulting in a payback period of 20 years. These payback times are less than the anticipated life expectancy of the solar ducts of 30 years.

As the existing air handling units that serve the main pool area are at end of life and in various states of failure, replacement and relocation options are considered. Due to structural and accessibility constraints, new structural mezzanines to support the replacement air handling equipment are recommended. These mezzanines would be located above the building's existing office spaces, stairwells or track areas and would be accessed independently of the existing ceiling level catwalks. The estimated cost for the proposed new mezzanines, air handling units, and associated equipment is \$3.2M (class 4 estimate) including installation.

Based on the findings of this study, KGS Group recommends the consideration of both solar ventilation air heating options on the basis of considerable GHG emission reduction and minimal maintenance requirements. KGS Group also recommends the replacement of the existing air handling units prior to the recommended solar opportunities installation as this equipment is end of line and replacement would ensure a complete, modern and fully functional system is provided.



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1.0 INTRODUCTION

The City of Winnipeg retained KGS Group to reassess the solar energy opportunities for the Pan Am Pool. The City intends to implement solar energy opportunities to reduce the facility's energy consumption; therefore, reducing its operating costs and Greenhouse Gas (GHG) emissions. This assessment also reviews the replacement of five existing air handling units that are at end of life and located inaccessibly. These five air handling units are candidates for solar energy opportunity implementation.

Background information on the Pan Am Pool facility, its existing mechanical systems, and on the previously conducted solar energy opportunities study are outlined in the background section below. A discussion of the updated solar energy opportunities and the details regarding the air handling unit's replacement follows. This study considers the mechanical, electrical and structural impacts of the installations. The energy provided by the solar installation and the resulting offset in energy costs and GHG emissions are reanalyzed in the context of 2022.

2.0 BACKGROUND

The Pan Am Pool was originally constructed in 1966 to host the Pan American Games in 1967. Many renovations and several additions have been completed at the facility since its original construction. An addition containing a lap pool and children's (kiddie) pool was built in 1994 at the east end of the original pool area. This addition is generally referred to as the "Lap Pool Addition". Parts of the original pool area's east wall were removed for the lap pool addition. The lap pool addition is open to the main pool area at the pool deck and track levels; therefore, the main pool area and the lap pool addition function as one building in practice.

As part of the above-mentioned renovations, many of the building's original mechanical and electrical systems have been replaced. Several recent mechanical renovations include the replacement of the facility's boiler plant, domestic water heaters, pool water heat exchangers, lap pool air handling units and boiler room combustion make up air units. Where possible, equipment has been configured to provide redundancy.

Currently, the building does not utilize solar, geothermal, or any other non-traditional source of energy. All energy is sourced from natural gas and electricity. A summary of the building's total natural gas consumption for the past eight years is provided in Table 1 below.



Year	Natural Gas Consumption (m ³)
2014	781,905
2015	701,964
2016	661,568
2017	700,887
2018	679,075
2019	852,300
2020	728,649
2021	693,041

TABLE 1: TOTAL ANNUAL NATURAL GAS CONSUMPTION

These values can provide a useful reference when evaluating the amount of energy provided by the various solar opportunities.

KGS Group conducted a study assessing solar energy opportunities at the Pan Am Pool in 2018 (final report issued January 2019). This study concluded that the building has good potential for implementation of solar technologies on its roof and south facing elevation. The study noted that the installation of solar ducts on the main pool area roof is not feasible due to the additional structural loads that the roof was not designed to handle; therefore, the study recommended the installation of a solar wall on the south wall of the main pool area to serve its air handling units, and the installation of solar ducts on the roof of the lap pool addition to serve its air handling units.

Solar domestic water heating and photovoltaic systems were investigated by the 2018 study and were ruled out due to incompatibilities with the current operation of the building's mechanical systems, long payback periods and increased maintenance costs. The final recommendation of the 2018 study was to implement ventilation air heating via a solar wall serving the main pool area and solar ducts serving the lap pool addition based on a reasonable payback period, a considerable reduction in GHGs and minimal maintenance requirements.

In addition to the study of solar energy opportunities, KGS Group is also reviewing the replacement of five air handling units serving the main pool area and the associated spectator areas, locker rooms, showers and ancillary spaces. Air handling units F-1 through F-5 are original to the building, at end of life and in various states of failure. These existing air handling units are located within the main pool area's ceiling space and are only accessible via the catwalk network that runs within the roof structure of the building. As part of the replacement of these units, relocation for greater ease of maintenance is to be considered.

Units F-1 through F-4 serve to condition the pool area and to provide ventilation to its occupants and spectators. F-1 and F-2 supply air onto the curtain walls at each end of the main pool area. F-3 and F-4 supply air to the spectator areas on each side of the main pool. Exhaust fans F-7 through F-10 operate in conjunction



with these four air handling units to ventilate and maintain the appropriate pressure within the main pool area. Reportedly, the outdoor air and return air mixing dampers within these 4 air handling units have failed and are now manually locked in the fully recirculating position by facility operation staff. As they are not supplying the correct amount of fresh air to the facility, this does not meet the ASHRAE 62.1 standard. Further to this, the reduced ventilation also means that there may be a buildup of chloramines from the pool, which will increase the rate of metal corrosion around the facility and in the air handling systems. The pneumatic controls serving each of the air handling units and exhaust fans are not operating correctly. New replacement air handling units and supporting equipment, and the existing exhaust fans are to be tied into the existing DDC system.

Unit F-5 operates in conjunction with exhaust fan F-15 to ventilate the locker/change rooms and other pool operation ancillary spaces. The unit's distribution ductwork also contains booster fans and hydronic reheat coils for better control of each zone that it serves. This unit has completely failed and has reportedly been out of operation for several years. Its distribution ductwork appears to have been partially disconnected, and it's KGS Group's understanding that the system currently relies on the booster fans and reheat coils to redistribute ambient air from the main pool area to the locker/change rooms and other pool area ancillary spaces. The pneumatic controls serving air handling unit F-5 are to be replaced by electronic controls tied into the existing DDC system.

In its current operating state, the existing main pool area HVAC system is not meeting the minimum ventilation requirements of the building code.

3.0 DISCUSSION

The implementation of solar ventilation air heating equipment on the building's large flat rooftop and south facing elevation are reviewed, building on the findings of the 2018 study completed by KGS Group. The energy and cost savings, GHG reduction, construction cost and payback are reassessed in the context of 2022 pricing and rates. Solar thermal energy installations are eligible for government incentives and grants. This report accounts for the Green Energy Equipment Tax Credit from the Manitoba Provincial Government, which is a credit of 10% of the capital cost of installation.

New air handling unit selections based on the updated ventilation rates discussed below are also reviewed, as are possible options for relocating the air handling units for greater accessibility and ease of maintenance. The replacement unit costs including installation and the provision of structure to support the units in their proposed new location is provided.

3.1 Solar Ventilation Air Heating

Details regarding the rationale behind the selection of a solar wall to supply pre-heated ventilation air to air handling units F-1 through F-5 and solar ductwork to supply pre-heated ventilation air to air handling units AHU-1 and AHU-2 can be found in the 2018 study report prepared by KGS Group. Explanations of the operation of the solar ducts and solar wall and considerations regarding their installation, operation and maintenance can also be found in the 2018 study report.



3.1.1 EXISTING AIR HANDLING SYSTEMS WITH SOLAR HEATING POTENTIAL

The existing HVAC systems within the building where solar ventilation pre-heating could potentially be applied include:

- Air handling units F-1 to F-4, which serve the main pool area;
- Air handling unit F-5, which serves locker/change rooms and ancillary spaces below the spectator areas; and
- Air handling units AHU-1 and AHU-2, which serve the lap pool addition.

Air handling units F-1 to F-4 are located within the ceiling space of the main pool area and are original to the building. The outdoor air connections are currently located on the rooftop and terminated with low silhouette intake hoods. Each unit is designed for a total airflow rate of 20,000 CFM (cubic feet per minute). Based on the original construction drawings units F-1 and F-2 are designed to provide up to 100% outdoor air at 20,000 CFM each, while units F-3 and F-4 are only designed to provide up to 50% outdoor air at 10,000 CFM each.

The calculated minimum ventilation rate for the pool, deck and spectator areas based on ASHRAE Standard 62.1 is 34,000 CFM. It is assumed that new air handling units would be sized to satisfy this calculated ventilation requirement to meet code, therefore; the replacement air handling units are to provide 8,500 CFM of ventilation air each.

Air handling unit F-5 is also located within the ceiling space of the main pool area and is original to the building. This unit also draws outdoor air from a rooftop mounted low silhouette intake hood. It's designed for a total flow rate of 13,000 CFM and up to 100% outdoor air during occupied hours. The distribution ductwork serving F-5 includes booster fans and reheat coils in various branch ducts to provide greater control over zone temperature.

Air handling units AHU-1 and AHU-2 are located at grade on raised platforms adjacent to the east end of the lap pool addition. Exterior supply and return air ductwork connect each air handling unit to distribution ductwork with in the lap pool area. The exterior ductwork enters/exits the building at the roof of the lap pool area and is routed along the building's exterior wall down to each unit. The design total and outdoor airflows for AHU-1 and AHU-2 are 16,000 CFM and 6,500 cfm each respectively. The outdoor airflow rate is provided continuously during occupied hours, and intermittently as required for dehumidification during unoccupied hours.

All air handling units considered for solar ventilation air heating have hydronic heating coils that are served by the building's central natural gas fired boiler plant. The amount of available ventilation air associated with each air handling system considered for solar pre-heating is summarized in the table below:



Air Handling Unit	Ventilation Rate (CFM)
F-1	8,500
F-2	8,500
F-3	8,500
F-4	8,500
F-5	13,000
AHU-1	6,500
AHU-2	6,500
Total	60,000

TABLE 2: VENTILATION RATES AVAILABLE FOR SOLAR PRE-HEAT

3.1.2 PROPOSED SYSTEMS

As per the conclusions and recommendations of the 2018 assessment conducted by KGS Group, two proposed solar ventilation air heating options are considered:

- Solar wall on the south face of the main pool area to serve air handling units F-1 to F-5.
- Solar ducts on the roof of the lap pool to serve AHU-1 and AHU-2.

Both options are similar with respect to operation, configuration and construction methods. In each case, ventilation air would be drawn through a solar collector to pre-heat it as much as possible prior to drawing it into each air handling unit via roof mounted insulated ductwork. The life expectancy of the solar wall systems are 30 years.

A summer bypass system would be included in the outdoor air ductwork serving each unit and would consist of a gooseneck intake hood tied into the main duct run from the associated solar collector to the respective air handling unit intake. Motorized dampers would be provided in each of the two duct branches, allowing for each unit to retain economizer (free cooling) operation during the times of year where it is required, bypassing the solar wall or ducts to prevent excess heating of outdoor air. Controls for the motorized dampers would be tied into the building's DDC system and would act in sequence with the associated air handling unit.

All ductwork between the solar collectors and air handling units would be insulated complete with aluminum jacketing to withstand the outdoor conditions. The ducts would be supported on the rooftop by sleepers.

3.1.2.1 Option 1 – Main Pool Area South Solar Wall

The solar wall would span across the majority of main pool area's south exterior wall, extending from it's base just above the main floor overhang up to the underside of the disused ventilation air openings (See Photo 1). This southern exterior elevation presents the best opportunity for the installation of a solar wall for this building because it is large, has no obstructions, and it has the most solar exposure. The available dimensions



for a solar wall installation result in a total area of approximately 16,640 ft². Based on a combined airflow rate of 47,000 CFM for all connected air handling units, the resulting airflow per unit surface area is 2.8 CFM/ft². This is within the industry standard best practice airflow range of 2-10 CFM/ft² and allows the wall to provide a relatively high temperature rise for incoming outdoor air.

Pre-heated air would be ducted up to roof level from the top of the solar wall. From there, distribution ductwork would connect to each associated air handling unit as described above.



Photo 1: Main Pool Area South Wall

The weight of the solar wall cannot be supported from the existing wall structure; therefore, a dedicated standalone structural support system would be required. The support system would be constructed upon piles at the base, with steel columns extending up to the underside of the overhanging wall. A steel beam would span across the columns to fully support the underside of the new solar wall which would also be affixed to the existing exterior pool area wall for additional stability.

The construction cost of this option is estimated to be \$700,000. The Provincial Government tax incentive is equal to 10% bringing the total capital cost down to an estimated \$630,000.

3.1.2.2 Option 2 - Lap Pool Addition Rooftop Solar Ducts

Two solar duct arrays would serve air handling units AHU-1 and AHU-2, with one array serving each unit. Each array would include seven 48-foot-long solar duct sections resulting in a total surface area of 1,344 ft² surface area for each array. Based on a total outdoor airflow rate of 6,500 CFM per array, the resulting airflow per unit surface area is 4.8 CFM/ft². This is within the industry standard best practice airflow range of 2-10 CFM/ft² and allows the wall to provide a relatively high temperature rise for incoming outdoor air.



Pre-heated air would be drawn from each array into the associated air handling unit's existing return air duct near where it exits the building at the addition's roof. The new rooftop solar ductwork would rest upon sleepers supported by the lap pool addition's existing roof structure.



Photo 2: Lap Pool Addition Roof

The construction cost of this option is estimated to be \$220,000. The Provincial Government tax incentive is equal to 10% bringing the total capital cost down to an estimated \$198,000.

The approximate locations of both solar ventilation air heating options and the associated ductwork is shown on Figure 1 in Appendix B.

3.1.3 ANALYSIS - SOLAR VENTILATION AIR HEATING

Analysis of the solar ventilation systems was completed using RETScreen software and electronic spreadsheets, utilizing local climactic data for Winnipeg.

For this analysis, the following assumptions have been made:

- Heat loss from insulated ductwork is negligible.
- The summer bypasses are utilized to limit the supply air temperature to 79°F. Energy that could be collected to increase the air temperature beyond this value is not included in the energy savings calculations.
- Operating hours for the pool are from 5:30 AM to 10:30 PM on weekdays, and 6:30 AM to 8:30 PM on weekends.
- The facility operates 365 days of the year.
- The cost of natural gas varies yearly as shown Appendix C Payback Analysis.
- The thermal efficiency of the boiler plant is constant at 88%, based on manufacturer data for a return temperature of 140°F and input of 50%.



• There is no shading of the solar wall or solar ducts by adjacent structures.

The results of the analysis are summarized in Table 3 below. The table shows the total amount of natural gas required to heat the ventilation air for the current configuration with no solar heating (baseline), and the amount required if each solar ventilation air heating was implemented.

TABLE 3: VENTILATION AIR HEATING ANNUAL NATURAL GAS CONSUMPTION

Option	Baseline (m³)	With Solar Option (m ³)	Reduction (m ³)
Option 1 – Solar Wall (F-1 to F-5)	263,858	193,961	69,897
Option 2 – Solar Duct (AHU-1 & AHU-2)	81,953	57,724	24,229

The energy consumption values were used to determine the costs, cost savings, and payback period, which are shown in Appendix C and summarized in Table 4. The construction costs shown factor in government incentives and grants.

TABLE 4: VENTILATION AIR HEATING PAYBACK ANALYSIS

Option	Estimated Construction Cost	Payback considering Increasing Gas and Carbon Tax Costs
Option 1	\$630,000	22 years
Option 2	\$198,000	20 years

Based on a GHG emission factor of 0.00196 tonnes CO_2/m^3 of natural gas, the greenhouse gas emissions and reductions were calculated for each option on an annual basis and are summarized in Table 5 below.

TABLE 5: VENTILATION AIR HEATING ANNUAL GREENHOUSE GAS EMISSIONS

Option	GHG Emissions -Baseline (tCO ₂)	GHG Emissions - with Solar Ventilation (tCO ₂)	GHG Reduction (tCO ₂)		
Option 1	517.2	380.2	137.0		
Option 2	160.6	113.1	47.5		

3.1.4 SUMMARY OF SOLAR OPTIONS

The solar options considered in the above sections are summarized below for comparison purposes. Based on the comparison, the main pool area solar wall option has the longer payback and the higher capital cost per



ton of greenhouse gas (GHG) reduction. The lap pool addition solar duct option has the higher payback and the lower capital cost per ton of greenhouse gas (GHG) reduction.

TABLE	6:	VENTIL	ATION	AIR	HEATING
		OPTION	SUMM	ARY	

Option Estimated Construction Cost		Payback considering Increasing Gas and Carbon Tax Costs (Years)	GHG Reduction (tCO ₂)	Construction Cost / tCO ₂ Reduction	
Option 1 Solar Wall – Main Pool Area	\$630,000	22	137	\$4,599	
Option 2 Solar Ducts – Lap Pool Addition	\$198,000	20	47.5	\$4,169	

3.2 Air Handling Units Replacement

This portion of the study focuses on the replacement of air handlers F-1, F-2, F-3, F-4, and F-5. These units are original to the building, are difficult to maintain due to their location, and are being evaluated for replacement at the City of Winnipeg's request.

3.2.1 EQUIPMENT LOCATIONS

A primary objective in the replacement of the existing five air handling units that serve the main pool area and its ancillary spaces is the improvement of maintenance access. KGS Group have reviewed several options with respect to the new air handling units' locations. Commentary relating to each considered option follows:

3.2.1.1 Option 1 – New Air Handling Units in Existing Unit Locations

Replacing the existing air handling units with new units installed in the same locations was quickly eliminated as an option due to the extremely limited space for installation and maintenance of the units. Complete pool area shutdown would likely be required to facilitate this option. Additionally, physically reaching the mezzanines upon which the current units are located is extremely inconvenient, less ergonomic, and less safe than the City of Winnipeg building operation staff would prefer.

3.2.1.2 Option 2 – New Air Handling Units on Main Pool Area Roof

Replacing the existing air handling units with new exterior units installed on the roof of the main pool area was also eliminated as an option due to insufficient structural capacity to accommodate the snow loading that exterior units would create and insufficient roof access to facilitate convenient maintenance of the units.

Snow clearing on the building's roof would be a concern, and additional roof access points would need to be provided above the locations of the existing units. The structural and access provisions required to facilitate option 2 would be costly, and the physical access concerns outlined in option 1 above would not be appreciably improved.

Additionally, the significant humidity in the pool air poses a freezing risk for any extended unit shutdown (maintenance, component failure, etc.). The frozen condensation could make restarting the air handling unit a



maintenance heavy process. To mitigate this risk and improve maintenance access, a service corridor within each unit would be required at a significant increase in unit capital cost.

3.2.1.3 Option 3 – New Air Handling Units at Grade Below the Track

Replacing the existing air handling units with new exterior units located at grade beneath the overhangs created by the track surrounding the main pool area was considered. This option was eliminated due to the extensive and expensive highly insulated ductwork that would be required to connect the new air handling units to the existing distribution ductwork in the main pool area's ceiling space. Similar pool air humidity related concerns to those expressed for option 2 apply to this option as well. To mitigate this risk and improve maintenance access, a service corridor within each unit would be required at a significant increase in unit capital cost.

3.2.1.4 Option 4 - New Air Handling Units on New Indoor Mezzanines

Replacing the existing air handling units with new interior units located on new mezzanines above existing meeting/office spaces, stairwells or parts of the track was considered. The proposed mezzanines would be constructed in a similar manner to the mezzanine built for the boiler combustion air make-up unit replacement project designed by KGS Group and constructed in 2018.

Locating the new units indoors on new mezzanines improves maintenance space and physical accessibility while eliminating the need for extensive additional insulated outdoor ductwork or expensive unit features/construction.

Replacement unit F-5 would be installed on the existing mezzanine that was constructed as part of the 2018 air handler F-6 replacement project.



Photo 3: Proposed Location For F-5 (Existing Mezzanine Above Meeting Room)



Replacement units F-2 and F-4 along with their associated replacement pump and heat exchanger would be installed on a new mezzanine of similar construction above the meeting/office spaces adjacent to the opposite stairwell.



Photo 4: Proposed Mezzanine Area For F-2 & F-4 (Above Existing Office Space)

Replacement units F-1 is suggested to be located above the existing track on a new mezzanine, while F-3 could be located on a new mezzanine in one of two possible locations:

- New extended mezzanine above the stairwell adjacent to the existing mezzanine next to F-6 and the proposed location for F-5, or
- New mezzanine over the track on the north side of the main pool area



Photo 5: Proposed Mezzanine Area For F-3 (Above Existing Stairwell)





Photo 6: Proposed Mezzanine Area For F-1 & Alternate Location for Unit F-3 (Above Existing Track)

The proposed locations of all new mezzanines are outlined on Figure 1 attached in Appendix B.

3.2.2 EQUIPMENT CONFIGURATIONS AND CAPACITIES

New air handling units are designed to match the total airflow capacities of the existing units while adjusting the heating performance to meet current code ventilation rates. Equipment data sheets for the proposed equipment selections are attached in Appendix A. To match the existing original design, heating coils are sized based on the following assumed design temperatures:

- Preheat Coil Outdoor Entering Air Temperature = -30°F
- Preheat Coil Leaving Air Temperature = 50°F
- Reheat Coil Mixed Air Entering Temperature = 50°F
- Reheat Coil Leaving Air Temperature = 80°F

Supporting equipment for each group of replacement air handling units has also been selected. Pumps and heat exchangers are required to convert the heating medium from hot water to hot glycol. This is necessary to prevent risk of freezing in coils that heat outdoor air directly. Equipment data sheets for new pumps and heat exchangers is also attached in Appendix A. To match the existing building systems operation, heating coils, heat exchangers and pumps are sized based on the following assumed fluid types and temperatures:

- Main Boiler Hot Water Loop Supply Temperature = 160°F
- Main Boiler Hot Water Loop Return Temperature = 120°F
- Local 50% Propylene Glycol Loop Supply Temperature = 140°F
- Local 50% Propylene Glycol Loop Return Temperature = 100°F



Based on the above conditions that are common to all selections, replacement air handling units, heat exchangers and pumps were selected with the following capacities:

- F-1 & F-2
 - 20,000 CFM @ 2" W.C. External Static Pressure
 - 8500 CFM Maximum Outdoor Air
- F-3 & F-4
 - 20,000 CFM @ 2.25" W.C. External Static Pressure
 - 8500 CFM Maximum Outdoor Air
- F-5
 - 13,000 CFM @ 1.75" W.C. External Static Pressure
 - 13,000 CFM Maximum Outdoor Air
- Heat Exchanger & Pump Serving F-1 & F-3
 - 2850 Mbh Total Capacity
 - 5 hp Pump, 160 GPM @ 39 feet
- Heat Exchanger & Pump Serving F-2 & F-4
 - 2850 Mbh Total Capacity
 - 5 hp Pump, 160 GPM @ 39 feet
- Heat Exchanger & Pump Serving F-5
 - 1450 Mbh Total Capacity
 - 3 hp Pump, 81 GPM @ 42 feet

3.2.3 ANTICIPATED AIR HANDLING UNIT REPLACEMENT COSTS

The effort to demolish the existing air handling units and install replacement air handling units is considerable. This is due to the size and capacity of the equipment locations and structural constraints of the existing building. The requirement for two new structural mezzanines has been included. Based on our analysis and review of equipment vendors quotes and previous project actual construction costs the class 4 (-30% to +60%) estimated cost of construction for the replacement air handling units, associated structures and equipment (not including the solar ventilation systems) is \$3.2M in 2022.

4.0 CONCLUSIONS

The following conclusions are made based on the findings of this report:

- The Pan Am Pool currently utilizes natural gas and electricity as its sole sources of energy and does not utilize any alternative sources of energy.
- The building has a good potential for implementation of solar technologies on the roof and south facing wall, which are large and unobstructed.
- Solar ventilation air heating could potentially be implemented on the south facing wall via the installation of a solar wall.
- Solar ventilation air heating could potentially be implemented on the lap pool addition roof via the installation of solar ducts.



- Existing air handling units F-1 to F-4 are beyond their life expectancy and require replacement. Air handling unit F-5 has failed and is abandoned in place.
- The existing air handling units are inaccessible for maintenance; therefore, new replacement units are to be installed in alternative locations offering improved accessibility. Several new interior mezzanines are proposed to achieve this.

5.0 RECOMMENDATIONS

Based on the findings of the study, KGS Group recommends the consideration of implementing solar ventilation air heating 3.1.2.1 Option 1 - Main Pool Area South Solar Wall, and 3.1.2.2 Option 2 - Lap Pool Addition Rooftop Solar Ducts. Although the simple payback period for each option is longer than owners would typically consider favourable, the implementation of these options is based upon a considerable reduction in GHGs and the minimal maintenance requirements of the solar wall and solar ducts.

KGS Group also recommends the installation of new replacement air handling units F-1 through F-5 and their associated pumps and heat exchangers on new mezzanines within the main pool area, based on the following benefits:

- Adequate ventilation to meet the minimum requirements of the building code.
- Reduced maintenance requirements and improved maintenance access.
- Improved temperature and humidity control of pool area.
- Improved energy efficiency with low temperature heating coils and operational controls.
- The ability to connect the air handlers into a building wide control system upgrade.



APPENDIX A

Equipment Datasheets





Trane Performance Climate Changer Air Handler

Unit Overview - F-4									
Application	Unit Sizo	I	External Dimensions	Weight					
Application	Unit Size	Height	Width	Length	Installed	Rigging			
Indoor unit	CSAA040	70.8 in	112.5 in	247.4 in	6301 lb	6170 lb			
Quantity of Shinning Continue			Largest Ship Split		Hogwigst Ship Split	Elevation			
Quantity of Shi	pping sections	Height	Width	Length	rieaviest Ship Split	Lievation			
4 piece(s)		70.8 in	70.8 in 112.5 in 73.8 in		2649 lb	0.00 ft			
	Supply Fai	n							

Airflow 200

20000 cfm Total Static Pressure 4.106 in H2O

Construction FeaturesPanel2in. foam injected R-13
with thermal breakPanelAll unit inner panels -
stainless steelIntegral
Base Frame6in. integral base frameShort
Circuit
Current
Rating5 kAAgency
ApprovalUL listed unit



Unit Electrical				
Circuit	Voltage/Phase/Frequency	FLA	MCA	Max Fuse Size
Circuit number 1 Supply fan + controls-LL	575/3/60	22.52 A	28.02 A	50.00 A

Unit Controls	
Factory Controls Package	Variable volume
Controller Type	UC600
Controller mounting	Unit mounted
Controller location	Right

Warranty

Warranty section Std. warranty only

Air mixing section - Position: 1

Openings											
Face	Path	ı	Туре	A	Airflow Face Area Pressu Velocity Area Drop		Pressure Drop		Hood		
Тор	Outsid	de	Opposed blade damper	200	000 cfm 1228 ft/min		1	16.29 sq ft	0.126 in H2O		N/A
Filter											
Туре			Frame		MERV	Rating		Quantity			Size
Throwaway filter	s - MERV		2"		MERV 5			24.00 6.00		16in.x20in. 16in.x25in.	
Pressure D	Drop		Condition		Face V	elocity		Airflow			Area
0.507	0.507 Mid-life 143 ft/min 10000 cfm 70.00 sc					0.00 sq ft					
Section Options											
	Door Location Right										





Н	eating	coil	sect	ion -	Posit	ion: 2

Coil Con	struction	Coil Perf	ormance
Model	Hot water - 5/8" Shipping Coil, General (5W)	Сара	acity
Rows	1	Total	737.46 MBh
Tube Diameter	5/8in. tube diameter (15.875 mm)	А	ir
Coil Connection	Standard	Flow	8500 cfm
Tube Matl/Wall Thickness	.020" (0.508 mm) copper tubes	Entering Dry Bulb	-30.00 F
Fin Spacing	156 Per Foot	Leaving Dry Bulb	50.00 F
Fin Material	Aluminum fins	Pressure Drop	0.031 in H2O
Fin Type	Prima flo E (energy efficient)	Face Velocity	227 ft/min
Face Area 37.50 sq ft		Flu	uid
Coil (top/single) H x L	54 in. (1372 mm) X 100" (2540 mm) finned length	Flow	42.18 gpm
Casing	Stainless steel	Entering	140.00 F
Turbulators	Yes	Leaving	100.00 F
Rigging Weight	179.5 lb	Pressure Drop	6.96 ft fluid
Installed Weight	227.0 lb	Tube Velocity	2.59 ft/s
Coil Section	on Ontions	Reynolds Number	5587.52
		Туре	Propylene glycol
Extended Drain and Vent	Holes only	Concentration	50.00 %
Drain Pan	Stainless steel	Fouling Factor	0.00025 hr-sq ft-deg F/Btu
Drain Pan Size	Small	Volume	5.71 gal
Drain Connection Left		AHRI 410 C	lassification
Minimum Trap Height (L) 6.324 in		AHPL/10 Classification	NOT Certified by AHRI
H Trap Dimension	3.382 in	Data Generation Data	11/16/2022
J Trap Dimension	1.691 in		11/10/2022
Door Location	Right	number	2650

Note: Coil is NOT certified by AHRI. Coil is within the scope of AHRI Standard 410.

Air mixing section - Position: 3 Openings Face Velocity Pressure Drop Airflow Path Hood Face Туре Area Opposed blade damper 1228 ft/min 0.126 in H2O N/A Тор Return 20000 cfm 16.29 sq ft **Section Options**

Door Location Right

Filter section - Position: 4												
	Primary Filter											
Туре	Frame	Loading	Airflow	Face Area	Face Velocity	Condition	Pressure Drop	Filter Quantity	Filter Size			
Pleated media - MERV 8	2in. filter frame	Side load filters	20000 cfm	70.00 sq ft	286 ft/min	Mid-life	0.569 in H2O	24.00 6.00	16x20 16x25			
Filter Section Options												
	Door Location Right											





Heatin	a coi	sect	ion - I	Posit	tion: 5

Coil Con	struction	Coil Performance		
Model	Hot water - 1/2" Unit Optimized, General (UW)	Сара	acity	
Rows	2	Total	650.70 MBh	
Tube Diameter	1/2in. tube diameter (12.7 mm)	A	ir	
Coil Connection	Standard	Flow	20000 cfm	
Tube Matl/Wall Thickness	.016" (0.406 mm) copper tubes	Entering Dry Bulb	50.00 F	
Fin Spacing	124 Per Foot	Leaving Dry Bulb	80.00 F	
Fin Material	Aluminum fins	Pressure Drop	0.165 in H2O	
Fin Type	Delta flo E (energy efficient)	Face Velocity	501 ft/min	
Face Area 39.93 sq ft		Flu	uid	
Coil (top/single) H x L	57 in. (1448 mm) X 100" (2540 mm) finned length	Flow	37.22 gpm	
Casing	Stainless steel	Entering	140.00 F	
Turbulators	Yes	Leaving	100.00 F	
Rigging Weight	246.2 lb	Pressure Drop	2.35 ft fluid	
Installed Weight	330.0 lb	Tube Velocity	1.42 ft/s	
Coil Section	on Options	Reynolds Number	2457.00	
Extended Drain and Vent		Туре	Propylene glycol	
Extended Drain and Vent	Holes only	Concentration	50.00 %	
Drain Pan Drain Dan Siza	Stainless steel	Fouling Factor	0.00025 hr-sq ft-deg F/Btu	
Drain Pari Size		Volume	10.06 gal	
Drain Connection Left		AHRI 410 C	lassification	
	0.200 III	AHRI 410 Classification	NOT Certified by AHRI	
		Data Generation Date	10/20/2022	
J Trap Dimension 2.337 in Door Location Right		Trane Select Assist update number	2640	

Note: Coil is NOT certified by AHRI. Coil is within the scope of AHRI Standard 410.

Supply fan section - Position: 6

oupply fail see		511. 0						
	Fan	Data			Motor	r Data	ı	
Wheel Diame	ter/Tyne/Class	24.5in. dd plenum,	full width, H		Power / Fan	10 h	p	
Wheel Blanc		press			Voltage	575/3	3	
	Fan Quantity	2			Speed	1800)	
Disc	harge Location	Top front			Class	NEM	IA premium comp	oliant TEFC
	Motor Location	Right side drive			Efficiency	92.12	2 %	
	Blades	Higher eff.(some ba	ands lower,more	Part Loa	ad Efficiency	89.40	0 %	
Drive	Service Factor	Direct drive		Fan electrical	power (FEP)	15.76	6 kW	
	Ean Porf	ormance		A	AHRI VFD HP	20.00	00 hp	
				Wire to air stat	tic efficiency	61.10	6 %	
	Airflow	20000 cfm		Note: VFD driven motor fai	n electrical power	calcula	ated in accordance w	ith AHRI 430.
Total Static Pressure 4.106 in H2O			Fan Section Options					
Tota	al Brake Power	18.637 np	_	Fan Wheel Balance Inverter balance with shaft grounding			shaft	
U	perating Speed	1756 rpm					onan	
	AMCA FEG	FEG85		Door Location Right				
Bare fan peak	total efficiency	76.70 %		Door Guard Yes				
Unit S	tatic Efficiency	69.46 %						
	Motor Interf	ace Options						
	Selection Type	VFD						
	Voltage	575/3						
Mou	Inting Location	Internal mounting						
Motor V	Vire In Conduit	Motor wiring condu	it					
١	/FD Frequency	60.00 Hz						
			Fan Discharg	ge Options				
Face	Туре	Airflow	Face Velocity	Area	Pressure D	rop	Exhaust Hood	Damper Torque Requirement
	Sizeable							

Note: Certified by the AHRI Central Station Air-Handling Unit (AHU) Certification Program, based on AHRI Standard 430/431. AHRI certified units are subject to rigorous and continuous testing, have performance ratings independently measured and are third party verified. Certified units may be found in the AHRI Directory at www.ahridirectory.org.

14.70 sq ft

1361 ft/min

0.231 in H2O

N/A



Top Face Feature

rectangular opening 20000 cfm

N/A



CSAA Quantity: 1 Tags: F-4

Pressure Drop in (in w.g.)						
Supply fan						
Air mixing section	0.63					
Coil section	0.13					
Air mixing section	0.13					
Filter section	0.57					
Coil section	0.17					
Fan section	0.23					
Internal Static Pressure	1.86					
External Static Pressure	2.25					
Total Static Pressure	4.11					

Starter/VFD only section - Position: 7

Supply Fan Motor Interface Door Right

TRANE[®]



CSAA Quantity: 1 Tags: F-4

2022/11/16 12:53:05 Product Version: 1

Product group: Indoor unit

Unit size: 40

Paint:





Trane Performance Climate Changer Air Handler

Unit Overview - F-5										
Application	Unit Sizo	I	External Dimension	S	Weig	ght				
Application	Unit Size	Height	Width	Length	Installed	Rigging				
Indoor unit	CSAA021	56.3 in	80.0 in	165.1 in	3119 lb	3016 lb				
Quantity of Shinning Costions					Hogwiggt Ship Split	Elevation				
Quantity of Shi	pping sections	Height	Width	Length	neaviest Ship Split	Elevation				
3 pie	ce(s)	56.3 in	80.0 in	88.5 in	1817 lb	0.00 ft				
	Supply Fa	n								

Airflow 13000 cfm Total Static Pressure 4.678 in H2O

Construction Features 2in. foam injected R-13 with thermal break Panel All unit inner panels -Panel Material stainless steel Integral Base Frame 6in. integral base frame Short Circuit Current 5 kA Rating Agency Approval UL listed unit



Unit Electrical				
Circuit	Voltage/Phase/Frequency	FLA	MCA	Max Fuse Size
Circuit number 1 Supply fan + controls-LL	575/3/60	22.52 A	28.02 A	50.00 A

Unit Controls	
Factory Controls Package	Variable volume
Controller Type	UC600
Controller mounting	Unit mounted
Controller location	Right

Warranty

Warranty section Std. warranty only

Air mixing section - Position: 1												
	Openings											
Face	Path	Туре	Airflow	Face Velocity	Area	Pressure Drop	Hood					
Тор	Outside	High velocity opposed damper	13000 cfm	2486 ft/min	5.23 sq ft	1.115 in H2O	N/A					
Section Options												
	Door Location Right											





Air mixing section - Position: 2

Openings												
Face	Patl	n	Туре	Airflow	Face Velocity		Area	Pre	essure Drop	Hood		
Тор	Retu	rn	Opposed blade damper	er 13000 cfm 1505 ft/min 8.64 sq ft		9.64 sq ft	0.24	6 in H2O	N/A			
	Filter											
Туре			Frame	MERV	Rating		Quantity			Size		
Pleated media -	MERV 8		2"	ME	RV 8	12.00			16	6in.x25in.		
Pressure Drop			Condition	Face V	Face Velocity		Airflow			Area		
0.607 Mid-life 390 ft/min 13000					13000 cfm		3	3.33 sq ft				
	Section Options											

Door Location Right

Heating coil section - Position: 3

.							
Coil Con	struction	Coil Performance					
Model	Hot water - 5/8" Shipping Coil, General (W)	Сара	acity				
Rows	4	Total	1409.85 MBh				
Tube Diameter	5/8in. tube diameter (15.875 mm)	A	lir				
Coil Connection	Standard	13000 cfm					
Tube Matl/Wall Thickness	.020" (0.508 mm) copper tubes	Entering Dry Bulb	-30.00 F				
Fin Spacing	107 Per Foot	Leaving Dry Bulb	70.00 F				
Fin Material	Aluminum fins	Pressure Drop	0.544 in H2O				
Fin Type	Prima flo H (Hi efficient)	Face Velocity	655 ft/min				
Face Area	19.83 sq ft	Flu	uid				
Coil (top/single) H x L	42 in. (1067 mm) X 68" (1727 mm) finned length	Flow	80.63 gpm				
Casing	Stainless steel	Entering	140.00 F				
Turbulators	Yes	Leaving	100.00 F				
Rigging Weight	302.8 lb	Pressure Drop	9.82 ft fluid				
Installed Weight	405.0 lb	Tube Velocity	3.18 ft/s				
Coil Section	on Options	Reynolds Number	6867.02				
Extended Drein and Vant		Туре	Propylene glycol				
Extended Drain and Vent		Concentration	50.00 %				
Drain Pan	Stainless steel	Fouling Factor	0.00025 hr-sq ft-deg F/Btu				
Drain Connection		Volume	12.28 gal				
Minimum Trap Height (L)	8.169 in	AHRI 410 C	lassification				
H Trap Dimension	4.779 in	AHRI 410 Classification	NOT Certified by AHRI				
J Trap Dimension	2.390 IN	Data Generation Date	10/25/2022				
		Trane Select Assist update number	2640				

Note: Coil is NOT certified by AHRI. Coil is within the scope of AHRI Standard 410.

Access/blank/turning section - Position: 4									
Options									
Section Length	14.000 in								
Door Location 1	Right								





Supply fail sec		511. 5											
	Fan	Data		Motor Data									
Wheel Diame	eter/Type/Class	24.5in. dd plenum,	80% width, H		Power / Fan	20 hp							
Wheel Blank		press			Voltage	575/3							
	Fan Quantity	1			Speed	1800	1800						
Disc	harge Location	Top front			Class	NEMA premium compliant TEFC							
	Motor Location	Right side drive			Efficiency	93.07 %							
	Blades	Higher eff.(some ba	ands lower,more	Part Loa	ad Efficiency	89.46 %							
Drive	Service Factor	Direct drive		Fan electrical	power (FEP)	13.51	51 kW						
	Ean Porf	ormance		I	AHRI VFD HP	20.00)0 hp						
				Wire to air sta	tic efficiency	52.83	3 %						
	Airflow	13000 cfm		Note: VFD driven motor fa	n electrical power	calcula	ted in accordance w	ith AHRI 430.					
Total	Static Pressure	4.678 IN H2O			Fan Sectio	on Opt	tions						
101	nerating Speed	2331 rpm		Ean W	haal Balanaa	Inverter balance with shaft							
J	Total Brake HP	16 350 hp			grounding								
		FEG85		D	oor Location	Right							
Bare fan neak	total efficiency	76 70 %			Door Guard	Yes							
Unit S	tatic Efficiency	58 63 %											
onin o	Motor Interf	ace Ontions											
	Selection Type												
Mai	voitage	575/3											
IVIOL Matar V	Mine In Conduit	Meter wiring	:.										
INIOTOR N	VIRE IN CONduit		π										
	VFD Frequency	79.00 HZ											
			Fan Discha	rge Options									
Face	Туре	Airflow	Face Velocity	Area	Pressure D	rop	Exhaust Hood	Damper Torque Requiremen					
Top Face Feature	Sizeable rectangular opening	13000 cfm	1826 ft/min	7.12 sq ft	0.417 in H2O		N/A	N/A					

Note: Certified by the AHRI Central Station Air-Handling Unit (AHU) Certification Program, based on AHRI Standard 430/431. AHRI certified units are subject to rigorous and continuous testing, have performance ratings independently measured and are third party verified. Certified units may be found in the AHRI Directory at www.ahridirectory.org.

CERTIFIED® www.ahridirectory.org

Pressure Drop in (in w.g.)										
Supply fan										
Air mixing section	1.11									
Air mixing section	0.85									
Coil section	0.54									
Fan section	0.42									
Internal Static Pressure	2.93									
External Static Pressure	1.75									
Total Static Pressure	4.68									

Starter/VFD only section - Position: 6

Supply Fan Motor Interface Door Right

TRANE[®]



CSAA Quantity: 1 Tags: F-5

Tags: F-5

Rigging weight: 3016.5 / Installed weight: 3118.7

Performance Climate Changer

Air Handlers

Product group: Indoor unit

Unit size: 21

Paint:







For maneuvering purposes, include 1.125 inches to each ship split length for overlapping panel flange. Flange will not add to overall installed unit length sh

Pos #	Module	Length	Weight
1	Air mixing section	28 1/2	335.25
2	Air mixing section	34 1/8	439.78
3	Coil section	14	526.38
4	Access section	14	136.06
5	Fan section	50 5/8	1315.30
6	Controls section	23 7/8	365.95
	Installed U	nit Weigh	t 3118.72 lbs



APPENDIX B

Figures



SCALE: 1:250mm (24"x36") NOTES:

THESE DRAWINGS SHALL NOT BE SCALED.

THE CONTRACTOR SHALL VISIT THE SITE AND SATISFY ONESELF ALL DIMENSIONS, DATUM, AND DETAILED INFORMATION SHOWN ARE CORRECT.

THE CONTRACTOR IS TO REVIEW AND COORDINATE ALL ARCHITECTURAL, MECHANICAL, ELECTRICAL AND STRUCTURAL DRAWINGS FOR ADDITIONAL OPENINGS THROUGH FLOORS, WALLS, AND CEILINGS FOR DUCT, PIPE & ELECTRICAL RISERS AND ALL OPENINGS NOT SHOWN ON DRAWINGS.

ALL OPENINGS THROUGH THE FIRE SEPARATIONS AS A RESULT OF THE WORK OF THIS CONTRACT ARE TO BE FIRE STOPPED AND SEALED WITH ULC APPROVED FIRE STOPPING TO MAINTAIN THE INTEGRITY OF THE FIRE SEPARATION, AND PROVIDE AND SMOKE-TIGHT BARRIER.

ALL PRODUCTS AND MATERIALS TO BE USED AND INSTALLED SHALL CONFORM WITH MANUFACTURER'S SPECIFICATIONS & APPLICABLE CODES.

THE CONTRACTOR SHALL BE RESPONSIBLE TO PATCH AND MAKE GOOD ALL EXISTING CONSTRUCTION AFFECTED BY THE REMOVAL OF ALL ITEMS FORMING THE PART OF THE **RENOVATION WORK.**

WHERE NEW FLOORING AND BASE IS TO BE INSTALLED IN EXISTING AREAS (REFER TO FLOOR PLAN AND ROOM SCHEDULE) THE EXISTING FLOORING SURFACE AND BASE MUST BE REMOVED, UNLESS OTHERWISE NOTED. ALL FLOOR SURFACES SHALL BE PREPARED IN ACCORDANCE TO MANUFACTURER'S RECOMMENDATIONS FOR INSTALLATION OF NEW FLOOR.

WHERE PAINTING OF EXISTING WALLS IS INDICATED ON THE ROOM SCHEDULE, THESE WALLS MUST BE CLEANED OF ANY EXISTING WALL COVERING, PATCHED & PREPARED TO ACCEPT NEW MATERIAL, UNLESS OTHERWISE NOTED.







22-0107-012

AS SHOWN

GROUP

APPENDIX C

Payback Analysis

Utility Rate Costs																										
		Year	ar Average Rate Escalation (2%) Carbon Tax Charge Total								Gas Consumption per Year												Install Costs			
		2018	\$	0.12	\$	-	\$ 0.	02	\$ 0.14		Option 1	Baseline	263860) m3			Option 2	Baseline	8195	81952 m3		ion 1	\$	630,000.00		
		2019	\$	0.09	\$	-	\$ 0.	04	\$ 0.13		New 193960 m3) m3		Option 2		New 57720 m3		Opt	Option 2		198,000.00				
		2020	\$	0.08	\$	-	\$ 0.	06	\$ 0.14																	
		2021	\$	0.18	\$	-	\$ 0.	08	\$ 0.26				Option 1						<u>.</u>							
n	P/F @ 2%	2022	\$	0.18	\$	-	\$ 0.	10	\$ 0.28	Base	line Gas Cost	New Gas Cost	Savings	Pre	sent Worth	Ren	maining Payback	Baseline Gas Cost	New Gas Cost	Savings	Pres	ent Worth	Rem	aining Payback		
1	0.9804	2023			\$	0.19	\$ 0.	13	\$ 0.31	\$	81,796.60	\$ 60,127.60	\$21,669.00	\$	21,244.29	\$	608,755.71	\$ 25,405.12	\$ 17,893.20	\$ 7,511.92	\$	7,364.69	\$	190,635.31		
2	0.9612	2024			\$	0.19	\$ 0.	16	\$ 0.35	\$	92,351.00	\$ 67,886.00	\$24,465.00	\$	23,515.76	\$	585,239.95	\$ 28,683.20	\$ 20,202.00	\$ 8,481.20	\$	8,152.13	\$	182,483.18		
3	0.9423	2025			\$	0.19	\$ 0.	19	\$ 0.38	\$	100,266.80	\$ 73,704.80	\$26,562.00	\$	25,029.37	\$	560,210.58	\$ 31,141.76	\$ 21,933.60	\$ 9,208.16	\$	8,676.85	\$	173,806.34		
4	0.9238	2026			\$	0.20	\$ 0.	22	\$ 0.41	\$	108,182.60	\$ 79,523.60	\$28,659.00	\$	26,475.18	\$	533,735.40	\$ 33,600.32	\$ 23,665.20	\$ 9,935.12	\$	9,178.06	\$	164,628.27		
5	0.9057	2027			\$	0.20	\$ 0.	24	\$ 0.45	\$	118,737.00	\$ 87,282.00	\$31,455.00	\$	28,488.79	\$	505,246.60	\$ 36,878.40	\$ 25,974.00	\$10,904.40	\$	9,876.12	\$	154,752.16		
6	0.888	2028			\$	0.20	\$ 0.	27	\$ 0.48	\$	126,652.80	\$ 93,100.80	\$33,552.00	\$	29,794.18	\$	475,452.43	\$ 39,336.96	\$ 27,705.60	\$11,631.36	\$	10,328.65	\$	144,423.51		
7	0.8706	2029			\$	0.21	\$ 0.	30	\$ 0.51	\$	134,568.60	\$ 98,919.60	\$35,649.00	\$	31,036.02	\$	444,416.41	\$ 41,795.52	\$ 29,437.20	\$12,358.32	\$	10,759.15	\$	133,664.36		
8	0.8535	2030			\$	0.21	\$ 0.	33	\$ 0.55	\$	145,123.00	\$ 106,678.00	\$38,445.00	\$	32,812.81	\$	411,603.60	\$ 45,073.60	\$ 31,746.00	\$13,327.60	\$	11,375.11	\$	122,289.25		
9	0.8368	2031			\$	0.22	\$ 0.	33	\$ 0.55	\$	145,123.00	\$ 106,678.00	\$38,445.00	\$	32,170.78	\$	379,432.83	\$ 45,073.60	\$ 31,746.00	\$13,327.60	\$	11,152.54	\$	111,136.71		
10	0.8203	2032			\$	0.22	\$ 0.	33	\$ 0.55	\$	145,123.00	\$ 106,678.00	\$38,445.00	\$	31,536.43	\$	347,896.39	\$ 45,073.60	\$ 31,746.00	\$13,327.60	\$	10,932.63	\$	100,204.08		
11	0.8043	2033			\$	0.23	\$ 0.	33	\$ 0.56	\$	147,761.60	\$ 108,617.60	\$39,144.00	\$	31,483.52	\$	316,412.87	\$ 45,893.12	\$ 32,323.20	\$13,569.92	\$	10,914.29	\$	89,289.80		
12	0.7885	2034			\$	0.23	\$ 0.	33	\$ 0.56	\$	147,761.60	\$ 108,617.60	\$39,144.00	\$	30,865.04	\$	285,547.83	\$ 45,893.12	\$ 32,323.20	\$13,569.92	\$	10,699.88	\$	78,589.91		
13	0.773	2035			\$	0.24	\$ 0.	33	\$ 0.57	\$	150,400.20	\$ 110,557.20	\$39,843.00	\$	30,798.64	\$	254,749.19	\$ 46,712.64	\$ 32,900.40	\$13,812.24	\$	10,676.86	\$	67,913.05		
14	0.7579	2036			\$	0.24	\$ 0.	33	\$ 0.57	\$	150,400.20	\$ 110,557.20	\$39,843.00	\$	30,197.01	\$	224,552.18	\$ 46,712.64	\$ 32,900.40	\$13,812.24	\$	10,468.30	\$	57,444.76		
15	0.743	2037			\$	0.24	\$ 0.	33	\$ 0.58	\$	153,038.80	\$ 112,496.80	\$40,542.00	\$	30,122.71	\$	194,429.47	\$ 47,532.16	\$ 33,477.60	\$14,054.56	\$	10,442.54	\$	47,002.22		
16	0.7284	2038			\$	0.25	\$ 0.	33	\$ 0.58	\$	153,038.80	\$ 112,496.80	\$40,542.00	\$	29,530.79	\$	164,898.68	\$ 47,532.16	\$ 33,477.60	\$14,054.56	\$	10,237.34	\$	36,764.88		
17	0.7142	2039			\$	0.25	\$ 0.	33	\$ 0.59	\$	155,677.40	\$ 114,436.40	\$41,241.00	\$	29,454.32	\$	135,444.36	\$ 48,351.68	\$ 34,054.80	\$14,296.88	\$	10,210.83	\$	26,554.04		
18	0.7002	2040			\$	0.26	\$ 0.	33	\$ 0.59	\$	155,677.40	\$ 114,436.40	\$41,241.00	\$	28,876.95	\$	106,567.41	\$ 48,351.68	\$ 34,054.80	\$14,296.88	\$	10,010.68	\$	16,543.37		
19	0.6864	2041			\$	0.27	\$ 0.	33	\$ 0.60	\$	158,316.00	\$ 116,376.00	\$41,940.00	\$	28,787.62	\$	77,779.79	\$ 49,171.20	\$ 34,632.00	\$14,539.20	\$	9,979.71	\$	6,563.66		
20	0.6730	2042			\$	0.27	\$ 0.	33	\$ 0.60	\$	158,316.00	\$ 116,376.00	\$41,940.00	\$	28,225.62	\$	49,554.17	\$ 49,171.20	\$ 34,632.00	\$14,539.20	\$	9,784.88	-\$	3,221.22		
21	0.6598	2043			\$	0.28	\$ 0.	33	\$ 0.61	\$	160,954.60	\$ 118,315.60	\$42,639.00	\$	28,133.21	\$	21,420.96	\$ 49,990.72	\$ 35,209.20	\$14,781.52	\$	9,752.85	-\$	12,974.07		
22	0.6468	2044			\$	0.28	\$ 0.	33	\$ 0.61	\$	160,954.60	\$ 118,315.60	\$42,639.00	\$	27,578.91	-\$	6,157.94	\$ 49,990.72	\$ 35,209.20	\$14,781.52	\$	9,560.69	-\$	22,534.75		
23	0.6342	2045			\$	0.29	\$ 0.	33	\$ 0.62	\$	163,593.20	\$ 120,255.20	\$43,338.00	\$	27,484.96	-\$	33,642.90	\$ 50,810.24	\$ 35,786.40	\$15,023.84	\$	9,528.12	-\$	32,062.87		
24	0.6217	2046			\$	0.29	\$ 0.	33	\$ 0.63	\$	166,231.80	\$ 122,194.80	\$44,037.00	\$	27,377.80	-\$	61,020.71	\$ 51,629.76	\$ 36,363.60	\$15,266.16	\$	9,490.97	-\$	41,553.84		
25	0.6095	2047			\$	0.30	\$ 0.	33	\$ 0.63	\$	166,231.80	\$ 122,194.80	\$44,037.00	\$	26,840.55	-\$	87,861.26	\$ 51,629.76	\$ 36,363.60	\$15,266.16	\$	9,304.72	-\$	50,858.57		
												<u>РАҮе</u> 22 Ү	BACK ears]					<u>PAY</u> 20 Y	BACK /ears]					



Experience in Action